



**Soil Compaction, Soil Shearing and Fuel Consumption:
A Practical Tool for Desision-Making in Farming and Forestry**

Autors
E. Diserens, A. Battiato

Programming
R. Meier

* former TASC designation *Tyres/Track And Soil Compaction*

Outline

Part 1 - Soil compaction

- Introduction – Problem
- Valuation principle
- Topsoil hardness
- Contact pressure and contact surface
- Stress propagation
- Indicative values
- Stress limits
- Solutions

Part 2 - Slip, traction force, soil failure and fuel consumption

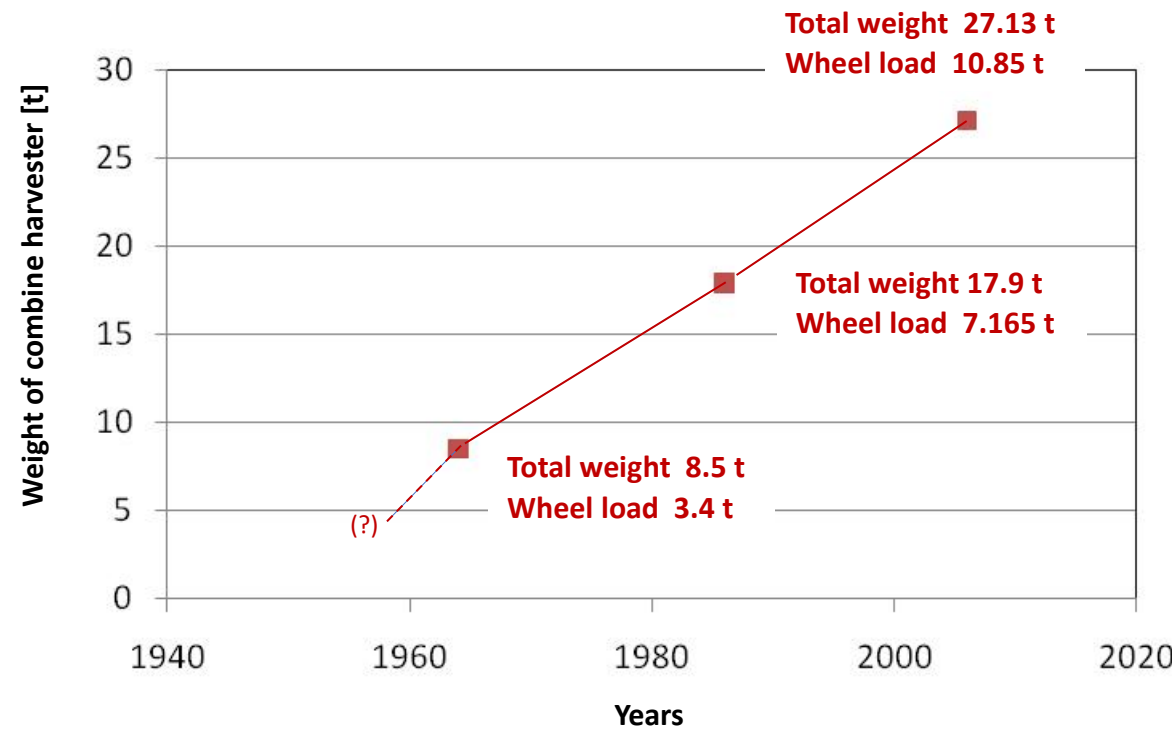
- Introduction – Problem
- Valuation principle
- Soil-tyres interaction model
- Soil shearing
- Traction-slip curve
- Traction force
- Fuel consumption
- Solutions

TASC – flow chart

Part 1 - Soil compaction



Size and power evolution of combine harvester



2006 – Claas Lexion 770
800/65R32 / 600/65R28
11.97 m (vario) / 261 kW



1986 – New Holland TX 34
30.5L32 / 16.70-20
5.18 m / 151 kW



1964 – Claas Matador Gigant
15-30AS
2.5 m / 88 kW



1960 –

Topsoil...



Source: L. Volk FH Soest

Ruts

Subsoil damages

Forestry



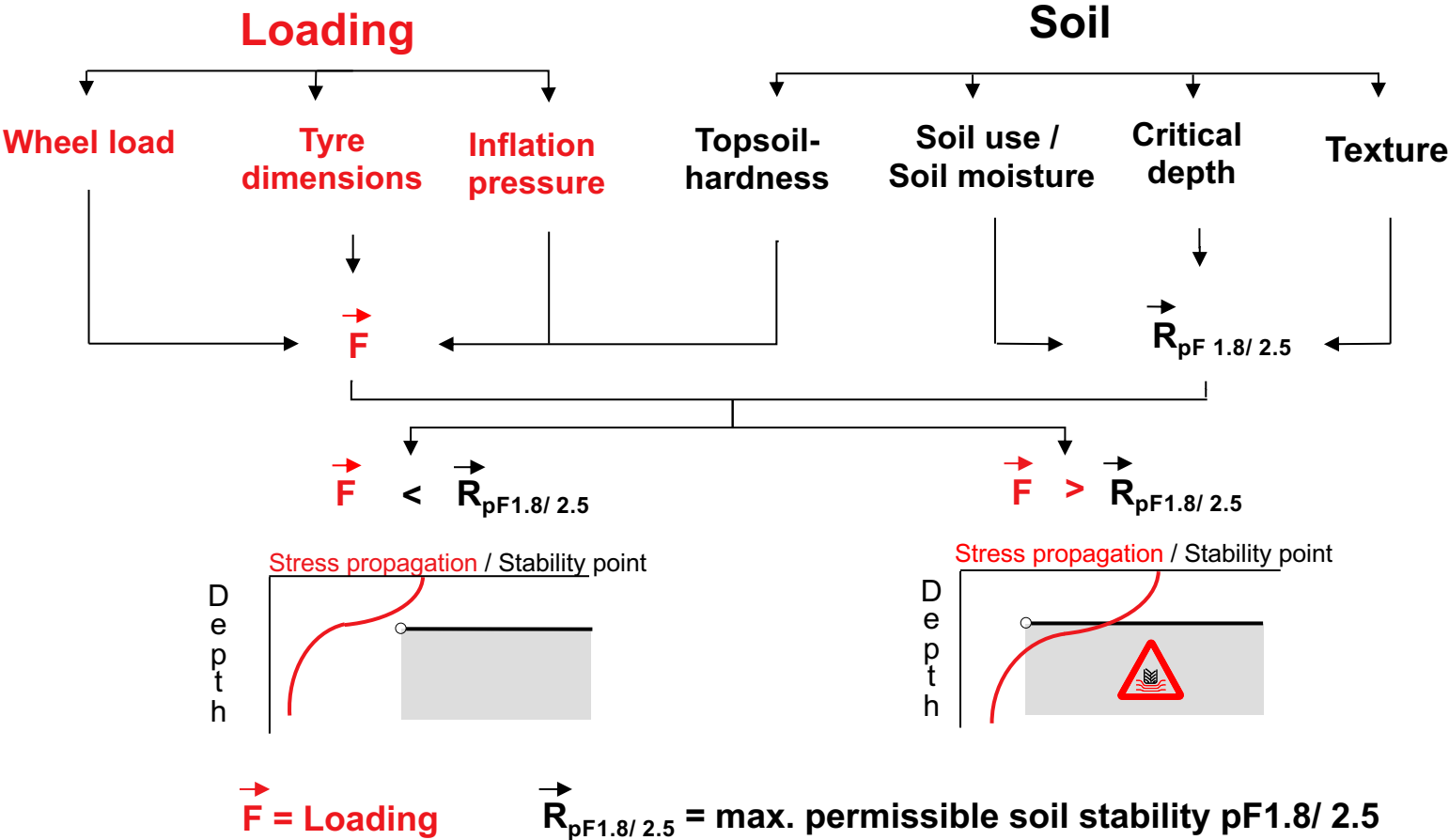
Source: WSL

Agriculture



Soil deformation in the depth
Asphyxiation

TASC - Basic principle - Compaction damage at the subsoil (below the critical depth)

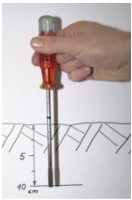
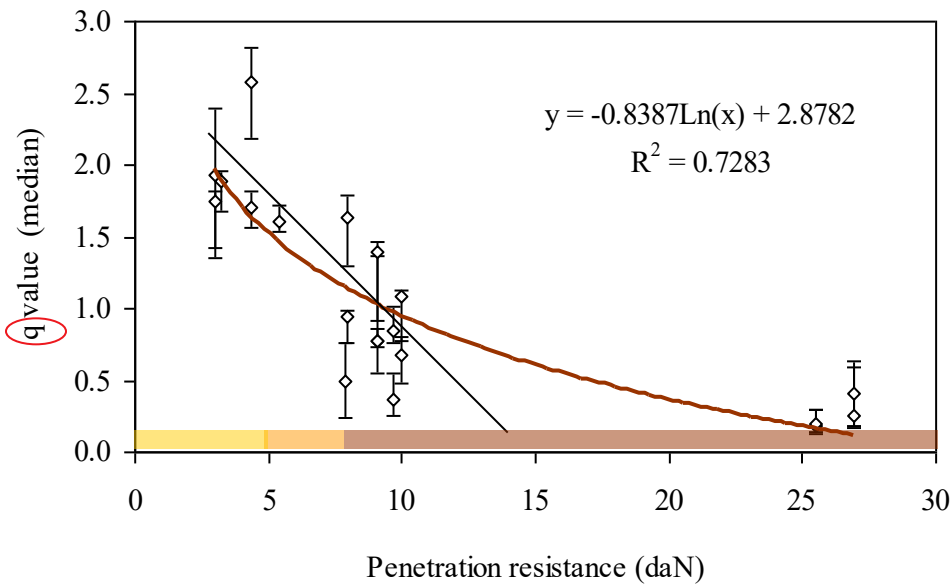


Topsoil hardness and vertical stress propagation σ_z

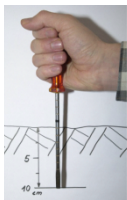
$$\sigma_z = \frac{2q\sigma_m}{\pi} \left[\arctan \frac{ab}{Rz} + \frac{abz}{R} \left(\frac{1}{a^2 + z^2} + \frac{1}{b^2 + z^2} \right) \right] \quad \text{with} \quad R^2 = a^2 + b^2 + z^2$$

according to Boussinesq (basic algorithm)

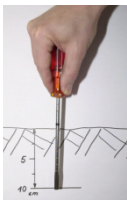
TASC V4.0.xls q: Factor for the topsoil hardness



1.73 = soft
39 N



1.26 = semi-firm
69 N



0.81 = firm
118 N

Screwdrivertest

(Soil&Tillage 102 (2009) 138-143)

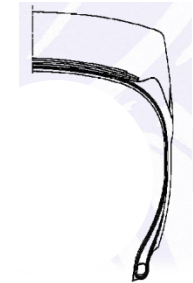
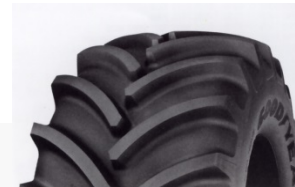
Contact pressure σ_m and contact surface A

$$\sigma_z = \frac{2q\sigma_m}{\pi} \left[\arctan \frac{ab}{Rz} + \frac{abz}{R} \left(\frac{1}{a^2 + z^2} + \frac{1}{b^2 + z^2} \right) \right] \quad \text{with} \quad R^2 = a^2 + b^2 + z^2$$

according to Boussinesq (basic algorithm)

Traction tyres

$$A [m^2] = f(b [m], d [m], F [kN], P_i [kPa])$$



Firm soil

$$b/d < 0.6 \quad A = 0.191 bd + 0.460 \cdot 10^{-3} F - 14.8 \cdot 10^{-5} P_i \quad n = 22; R^2 = 0.973; p_{BD} = 0.000; p_F = 0.004; p_{P_i} = 0.031$$

$$b/d \geq 0.6 \quad A = 0.187 bd + 0.382 \cdot 10^{-3} F - 28.4 \cdot 10^{-5} P_i \quad n = 42; R^2 = 0.950; p_{BD} = 0.000; p_F = 0.004; p_{P_i} = 0.069$$

Biosystems Engineering 110 (2011) 73-82

Soft soil

$$b/d < 0.6 \quad A = 0.247 BD + 0.582 \cdot 10^{-3} F - 19.3 \cdot 10^{-5} P_i \quad n = 7; R^2 = 0.949; p_{BD} = 0.109; p_F = 0.134; p_{P_i} = 0.536$$

$$b/d \geq 0.6 \quad A = 0.127 BD + 0.498 \cdot 10^{-3} F - 89.0 \cdot 10^{-5} P_i + 0.172 \quad n = 29; R^2 = 0.917; p_{BD} = 0.000; p_F = 0.000; p_{P_i} = 0.000$$

Encyclopedia of agrophysics, Springer, 2011

A: contact area; b: tyre width; d: tyre diameter; F: wheel weight; P_i: inflation pressure

Indicative values

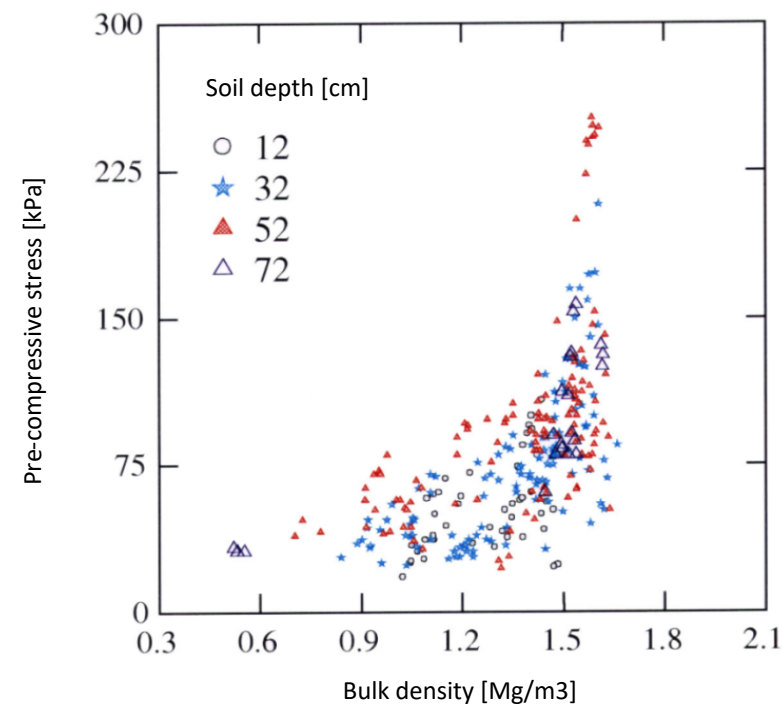
Effective bulk density δ_{eff}

$$\delta_{eff} = \delta_s + 0.009 \cdot C$$
 where: δ_s = bulk density [g/cm³] C = Clay content [%]

Indicative values δ_{eff}

for arable soil : 1.7 g/cm³
for forest soil : 1.5 g/cm³

Relationship pre-compressive stress
versus bulk density δ_s



Source: Quasem et al. 2000

Stress limits and texture under wet and dry soil conditions

| Classes of texture | Soil stability limit [kPa] Forest soil | | Soil stability limit [kPa] Agricultural soil | |
|----------------------------------|---|--------|---|--------|
| | pF 1.8 | pF 2.5 | pF 1.8 | pF 2.5 |
| Clay soil 45 % C | 55 | 70 | 80 | 95 |
| Sily soil 15 % C, 80 % U | 60 | 85 | 105 | 130 |
| Clay loam, loam 21 % C | 50 | 70 | 85 | 110 |
| Sandy loam, loamy sand 11 % C | 65 | 90 | 110 | 145 |
| Sand 5 % C | 75 | 110 | 130 | 170 |

Stress propagation and soil hardness - a clay and sand soil, semi-firm and firm



1964 – Claas Matador Gigant



1986 – New Holland TX 34



2006 – Claas Lexion 770

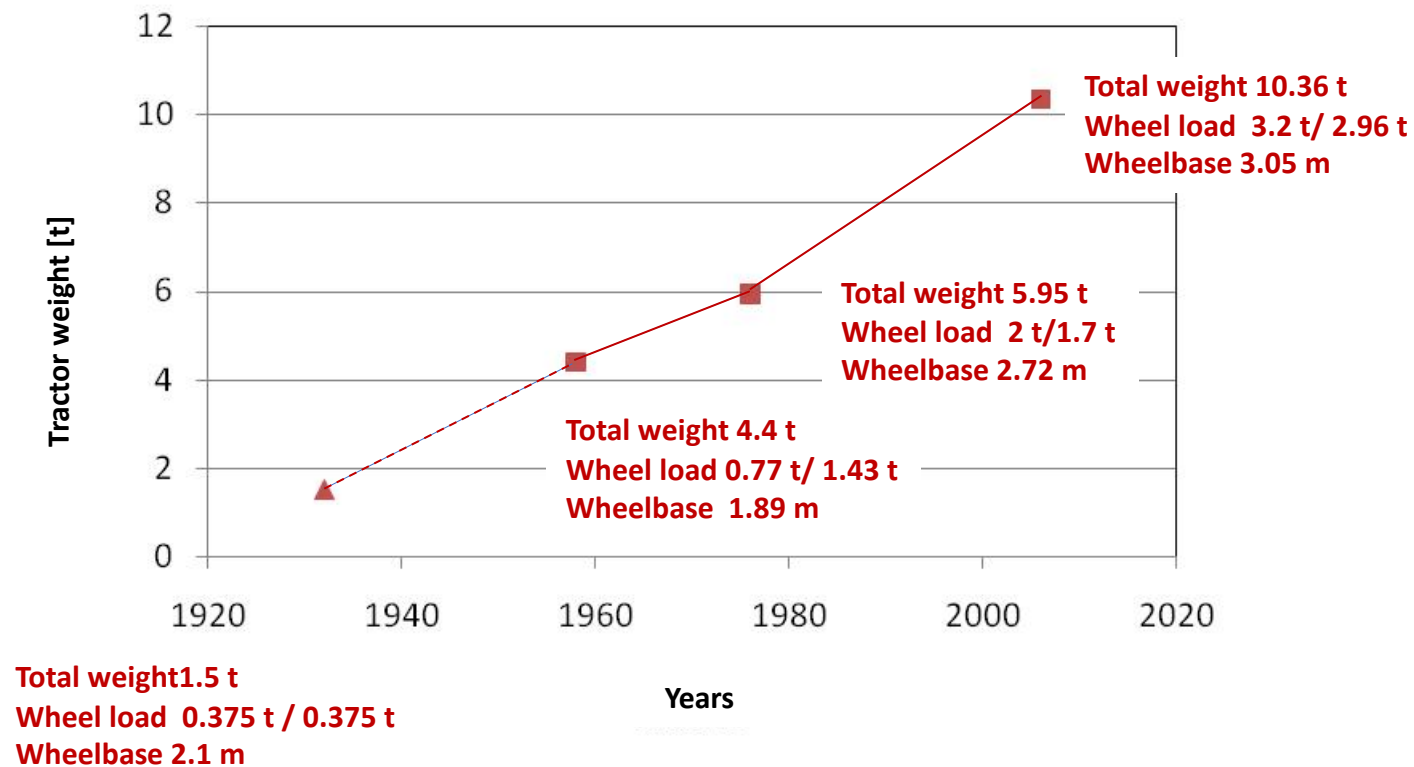
| | | | | | | | | | | | | |
|---------------------|----------------|------|------------|------|----------------|------|------------|------|----------------|------|------------|------|
| soil hardness | semi firm 69 N | | | | semi firm 69 N | | | | semi firm 69 N | | | |
| soil moisture | dry pF 2.5 | | wet pF 1.8 | | dry pF 2.5 | | wet pF 1.8 | | dry pF 2.5 | | wet pF 1.8 | |
| soill texture | clay | sand | clay | sand | clay | sand | clay | sand | clay | sand | clay | sand |
| soil pressure [kPa] | 115 | | | | 130 | | | | 160 | | | |
| critical depth [cm] | 19 | 0 | 24 | 8 | 33 | 0 | 40 | 18 | 46 | 18 | 53 | 32 |
| soil hardness | | | | | firm 118 N | | firm 137 N | | firm 147 N | | firm 167 N | |
| soil moisture | | | | | dry pF 2.5 | | wet pF 1.8 | | dry pF 2.5 | | wet pF 1.8 | |
| soil texture | | | | | clay | sand | clay | sand | clay | sand | clay | sand |
| soil pressure [kPa] | | | | | 144 | | | | 178 | | | |
| critical depth [cm] | | | | | 21 | 0 | 23 | 0 | 22 | 0 | 22 | 0 |

Max. tilling depth : 25 cm

Part 2 - Slip, traction force, soil failure and fuel consumption



Size and power evolution of the tractors



2006 - Fendt vario 936
600/65R34 / 710/70R42
100/80 kPa
261 kW



1976 - Fendt 614 SL
16.9R26 / 20.8R38
130/100 kPa
100 kW



1958 - John Deere 830
7.5-18 / 18.4-34
250/80 kPa
56 kW

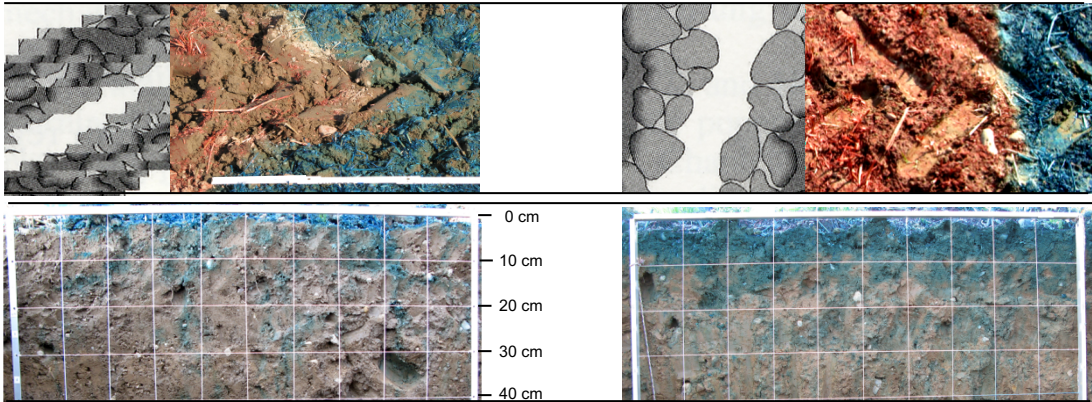


1936 - Massey-Harris 101S
5.5-16 / 11.2-24
100/100 kPa
23 kW

Impact on the topsoil



Impact on the subsoil

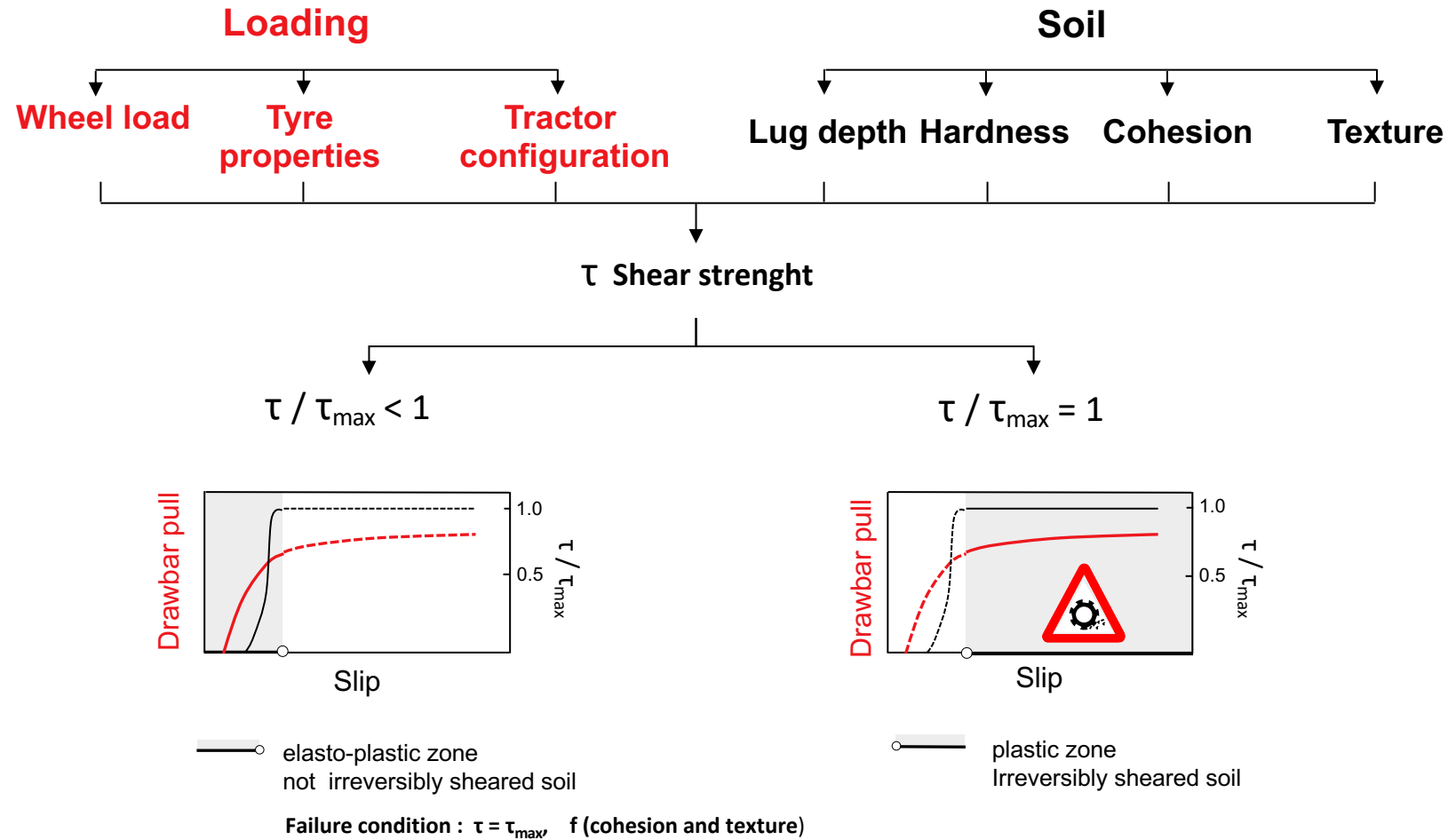


| Blue dye ** [Area distribution %] | | | Water infiltration ksat [10 ⁻² m/day] | | |
|-----------------------------------|-----------|----------|--|-----------|----------|
| Depth | 25 % slip | 0 % slip | Depth | 25 % slip | 0 % slip |
| 0.0 - 0.1 m | 21,84 | 81,00 | 0.0 - 0.1 m | 0.50 | 18.12 |
| 0.1 - 0.2 m | 15,10 | 49,02 | 0.1 - 0.2 m | 0.74 | 5.39 |
| 0.2 – 0.3 m | 8,56 | 40,08 | 0.2 – 0.3 m | - | - |
| 0.3 – 0.4 m | 4,74 | 22,48 | 0.3 – 0.4 m | 3.58 | 8.13 |

* Tractor H488 65 kW - 25 % slip

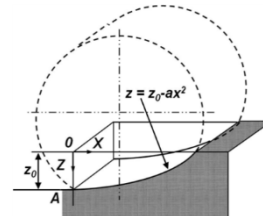
** stubble, loam soil

TASC - Basic principle - Shearing damage at the topsoil



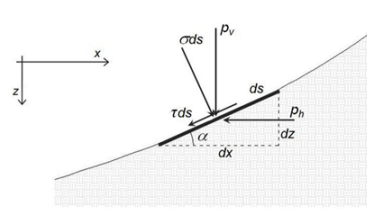
Soil-tyres interaction model

I – Soil-tyre contact surface z_0, a

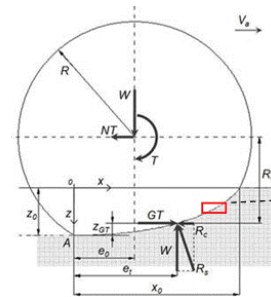


(Osetinsky & Shmulevich; 2004)

II – Stress distribution σ, τ along the contact surface



III – Gross traction GT , net traction NT , soil compaction resistance R_c , driving torque T and Power P



(Osetinsky A., Shmulevich I., 2004)

Soil-tyres interaction model

I – Soil-tyre contact surface z_0 , a

Friction modulus of deformation - K_ϕ and
cohesive modulus of deformation K_c of the soil

$$p_z = \left(\frac{K_c}{b} + K_\phi \right) z^n$$

Wheel load or total soil reaction

$$W = \frac{K_c + bK_\phi}{3} z_0^n \sqrt{\frac{z_0}{a}} (3-n)$$

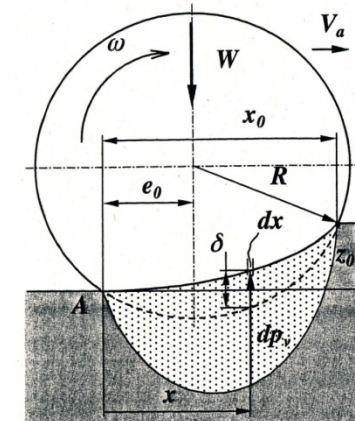
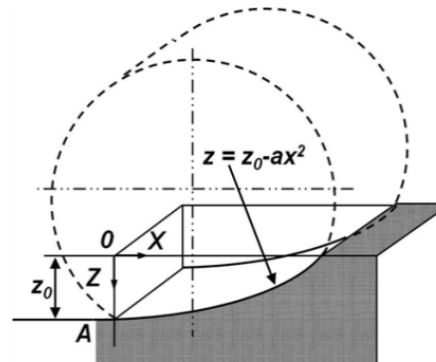
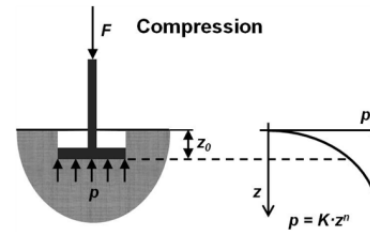
Wheel load from tyre parameters

$$W = \int_0^{x_0} K_v \delta dx$$

$$K_v = \frac{W}{R^2 \arcsin\left(\frac{x_0}{2R}\right) - \frac{x_0}{2} \left(R - \frac{W}{K_{cavc} + \Delta K_p P_i}\right)}$$

Implicite solution for z_0 and a

$$W = K_v \left\{ \frac{ax_0^3}{3} \sqrt{R^2 - e_0^2} \left(x_0 - \frac{e_0}{2} \right) + \frac{x_0 - e_0}{2} \sqrt{R^2 - (x_0 - e_0)^2} + \frac{R^2}{2} \left[\arcsin\left(\frac{x_0 - e_0}{R}\right) + \arcsin\left(\frac{e_0}{R}\right) \right] \right\}$$



(Osetinsky A., Shmulevich I., 2004)

Soil-tyres interaction model

II – Stress distribution along the contact surface, σ , τ and slip i , i_f , i_r

Vertical and horizontal components of the elementary force p_v and p_h

$$p_h = (\sigma ds) \sin \alpha - (\tau ds) \cos \alpha \quad \text{with} \quad ds \sin \alpha = b dz$$

$$p_v = (\sigma ds) \cos \alpha + (\tau ds) \sin \alpha \quad \text{with} \quad ds \cos \alpha = b dx$$

Shear stress τ , normal stress σ and soil displacement j

$$\tau = (c + \sigma \tan \varphi) (1 - e^{-j/k}) \quad \sigma = \frac{\sigma_v - c(1 - e^{-j/k}) 2ax}{1 + \tan \varphi (1 - e^{-j/k}) 2ax} \quad \text{with} \quad \sigma_v = \frac{dp_v}{b dx} = \frac{K_v \delta}{b}$$

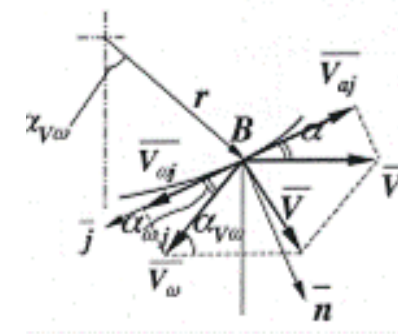
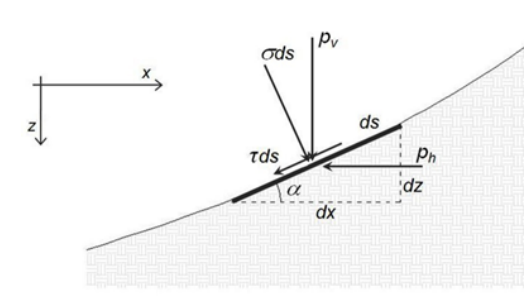
$$\text{with} \quad j = \int_0^x V_j dx \quad \text{with} \quad V_j = V_{aj} - V_{aj}$$

Slip i

$$i = \frac{\omega R_r - V_a}{\omega R_r} \quad \text{with} \quad R_r \text{ as rolling radius}$$

Slip of the front wheel- i_f slip of the rear wheel i_r

$$i_f = i_r$$



(Osetinsky A., Shmulevich I., 2004)

Soil-tyres interaction model

III – Gross traction GT , net traction NT , soil compaction resistance R_c , driving torque T and Power P

Gross traction GT

$$GT = \int_0^{x_0} p_h = \int_x^{x_0} \frac{\sigma_v [2\alpha x - \tan \varphi (1 - e^{-jfk})] - c(1 - e^{-jfk}) [(2\alpha x)^2 + 1]}{1 + 2\alpha x (1 - e^{-jfk}) \tan \varphi} b dx$$

Soil compaction resistance R_c

$$R_c = \int_0^{z_0} p_z b dz = (K_c + bK_s) \frac{z_0^{n+1}}{n+1}$$

Net traction NT

$$NT = GT - R_c$$

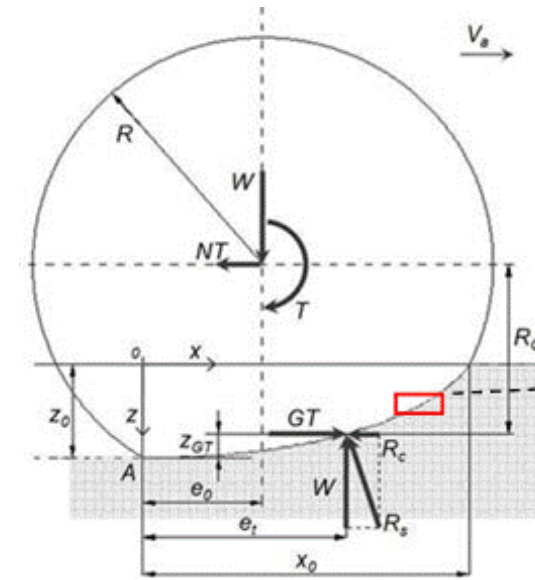
Driving torque T

$$M_r = W(e_t - e_0) \quad \text{with} \quad We_t = \int_0^{x_0} K_v \delta x dx$$

$$M_{GT} = GT \cdot R_{GT} \quad \text{with} \quad R_{GT} = \sqrt{R^2 - e_0^2} - z_{GT} \quad T = M_{GT} + M_r$$

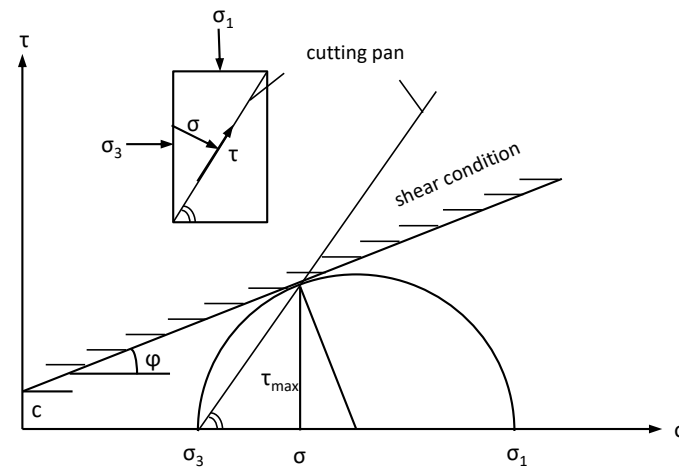
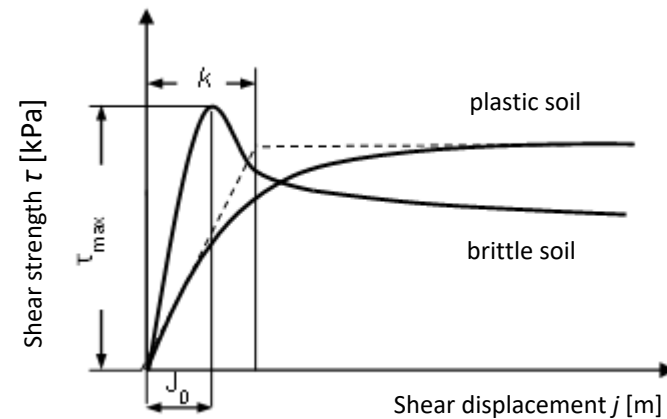
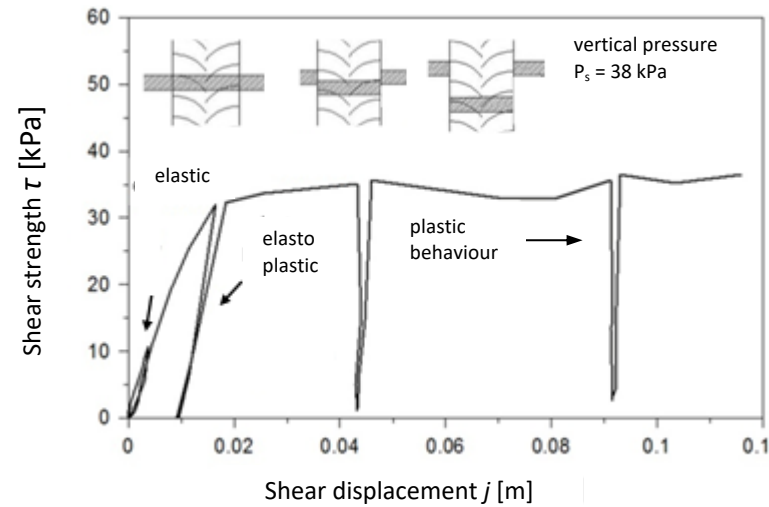
Net power P_{NT} , power on wheels P_{GT}

$$P_{NT} = NT \cdot V_a \quad P_{GT} = T \cdot \omega$$



(Osetinsky A., Shmulevich I., 2004)

Soil shearing - conditions



$$\tau_{max} = (c + \sigma \tan \phi)$$

$$\tau = (c + \sigma \tan \phi) (1 - e^{-j/k})$$

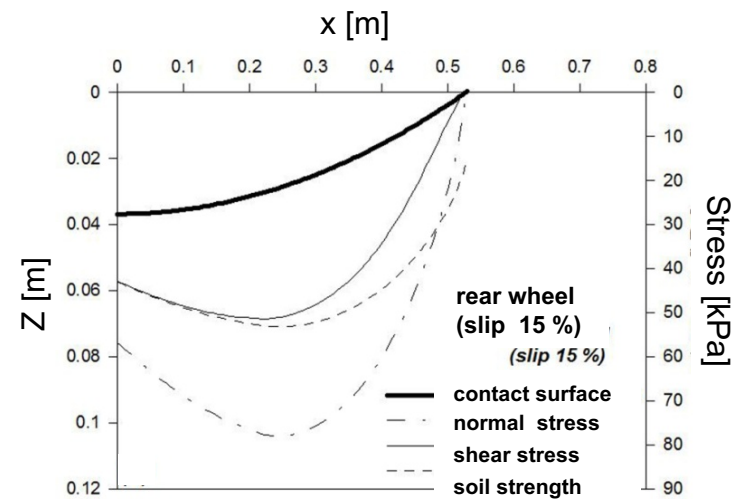
an approximation with:

$$\frac{\tau}{\tau_{max}} = \left(1 - e^{-\frac{j}{k}} \right) = 0.99$$

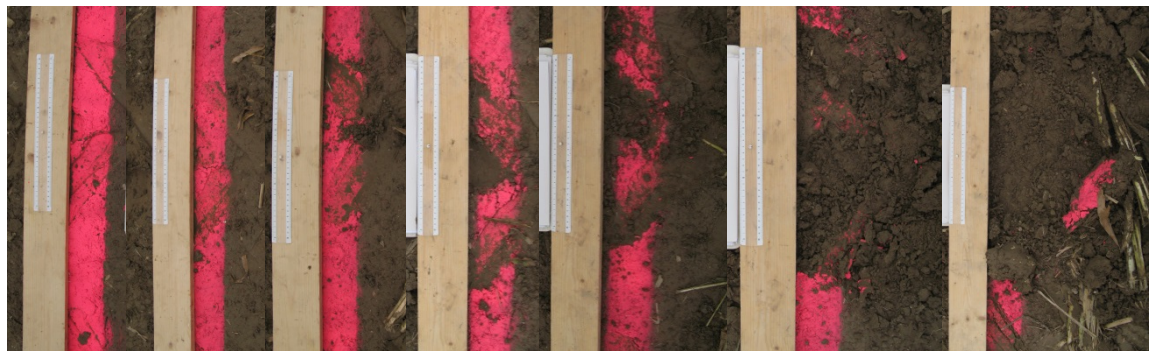
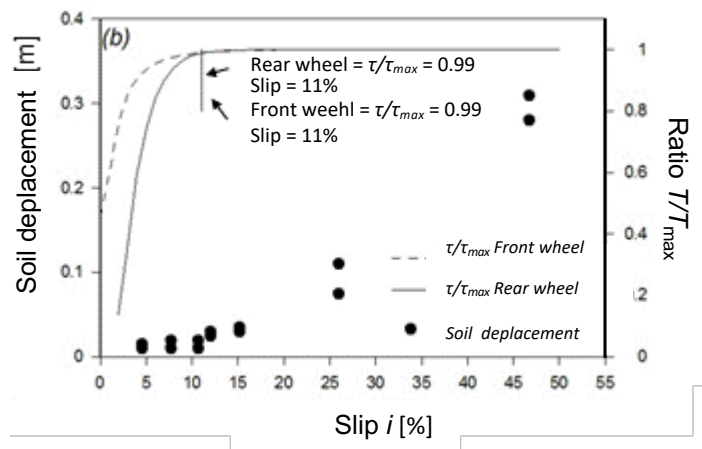
$$\frac{j}{k} = 4.6$$

Soil shearing – measurements and simulation

Tractor: 65 kW
 Total weight 4 t.
 Inflation pressure : 0.6 bar
 Soil: 27 % C; 53 % U; 20 % S



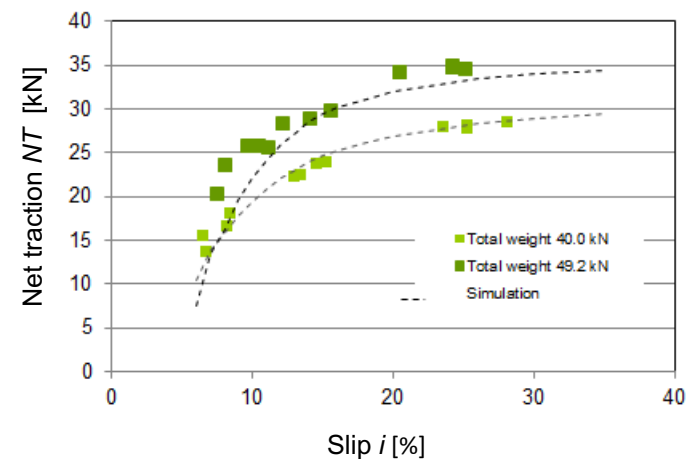
Soil shearing – measurements and simulation



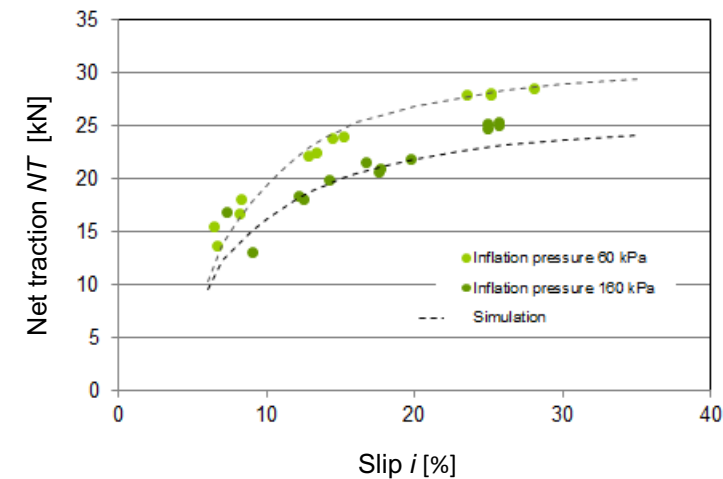
| | | | | | | | |
|------------------------|---|---|----|----|----|----|----|
| Slip [%] | 0 | 8 | 11 | 15 | 20 | 25 | 30 |
| Soil displacement [cm] | 0 | 1 | 2 | 3 | 4 | 11 | 28 |

Traction slip curves

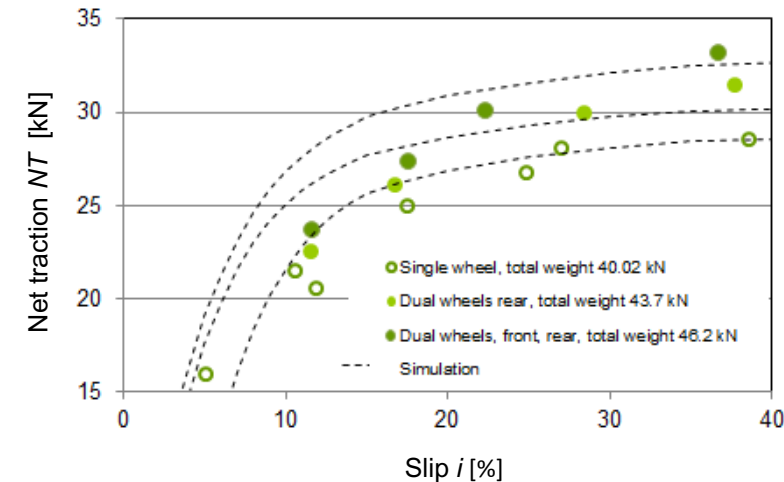
Influence of the total weight...



Influence of the inflation pressure...



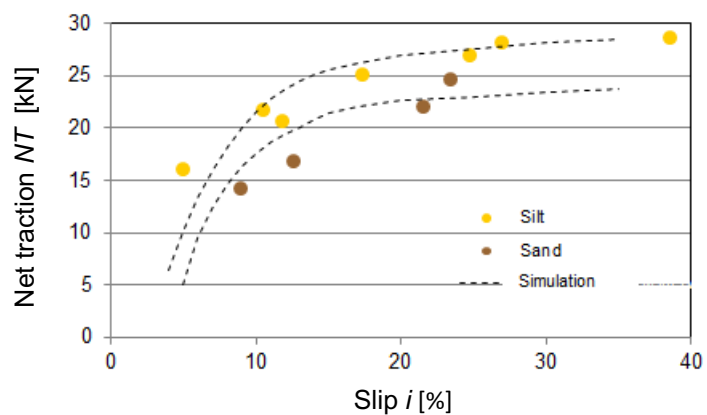
Influence of the dual wheels...



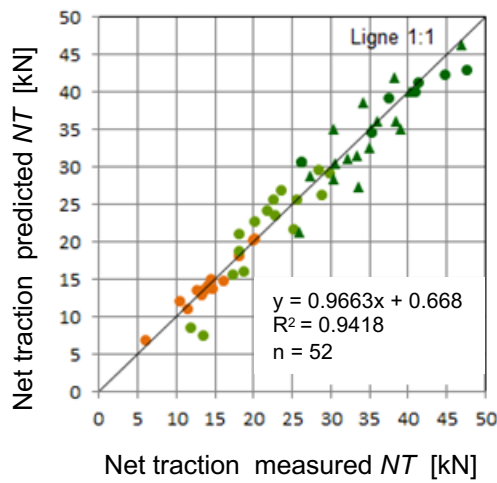
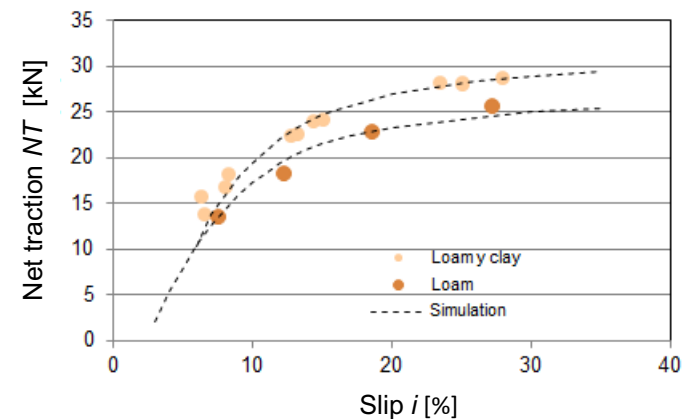
Traction slip curves

Influence of the soil texture

silt - sand



loamy clay – loam



Fiat 50-66 DTS, 40 kW
 John Deere 6920, 110 kW

Hürlimann DT, 65 kW
 John Deere 6930, 130 kW

Draft requirement

$$NT = F_i A = [A + Bv + Cv^2] \cdot WT$$

ASABE
American Society of Agricultural and Biological Engineers

with:

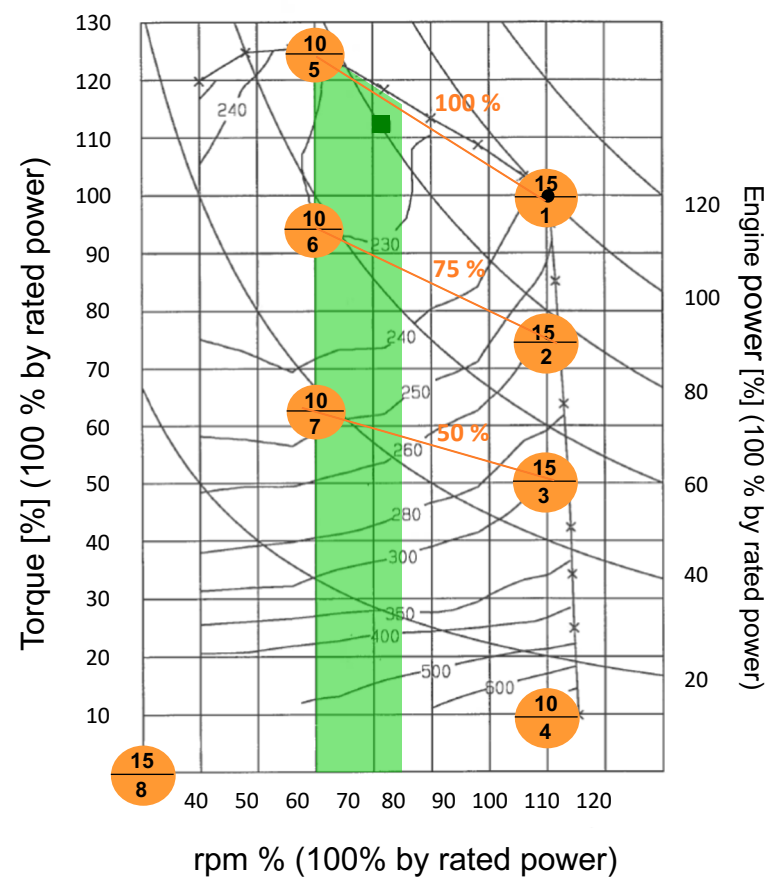
- D = Draft requirement [N]
- F = Soil parameters, depending on texture
- i = 1 for clay, 2 for loam and 3 for sand
- A, B and C = machine-specific parameters
- v = field speed [km/h]
- W = machine width [m] / number of tools [n_t] / number of rows [n_r]
- T = tillage depth [cm]



| Implements for soil tillage and seedi | | | Machine specific parameter | | | Soil specific parameter | | | |
|--|-------|------|----------------------------|-----|---|-------------------------|-----------------------------|-------------------------------|-------------------------------|
| | Width | unit | Units | A | B | C | F ₁ Fine soil | F ₂ Medium soil | F ₃ Coarse soil |
| SUBSOIL TILLAGE | | | | | | | | | |
| Subsoiler / Manure injector -- narrow point [t] | | | tools | 226 | 0 | 1.8 | 1 | 0.70 | 0.45 |
| Subsoiler / Manure injector - 0.3 m winged point [t] | | | tools | 294 | 0 | 2.4 | 1 | 0.70 | 0.45 |
| Moidboard Plow [m] | | m | | 652 | 0 | 5.1 | 1 | 0.70 | 0.45 |

➡ until 41 tools for subsoil tillage, topsoil tillage, seedlings and cultivation

Specific fuel consumption (engine)



- 100 % rated power
 - lowest specific fuel consumption
 - $\frac{15}{8}$ weighting %
measuring point
- ISO 8178 C1

| Tractors power classes | b_{eM} [g/kWh] |
|------------------------|------------------|
| ≤ 75 kW | 248 |
| >75 kW | 223 |

Schäffeler U., Keller M., 2008

Calculation of the specific fuel consumption

Engine power P_M [kW]

$$P_M = \frac{P_{GT}}{\eta_G}$$

P_{GT} : tractive power on the driving wheels [kW]

η_G : efficiency faktor for the transmission **0.85**

Hourly fuel consumption B_e [l/h]

$$B_e = P_M \cdot \frac{b_{eM}}{1000} \cdot \frac{1}{\delta_D}$$

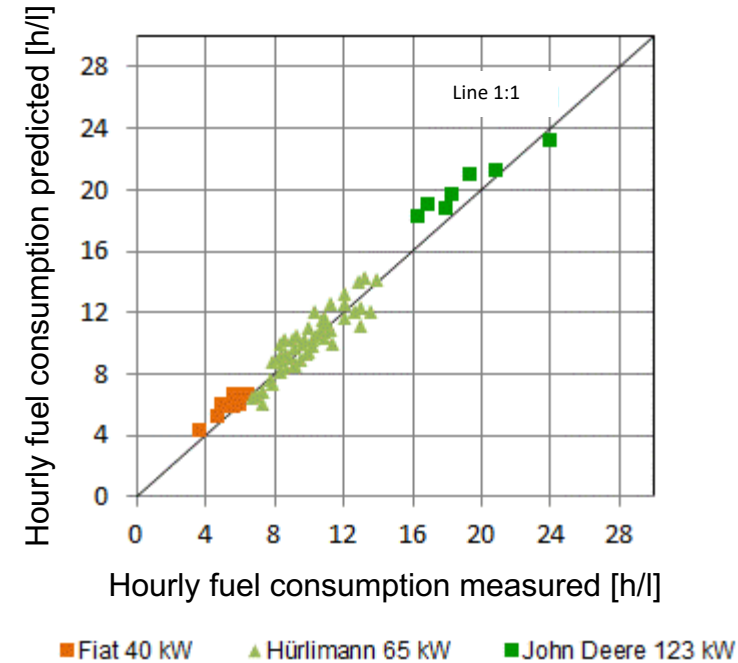
b_{eM} : specific fuel consumption of the engine [g/kWh]

δ_D : specific weight of the diesel [Mg/m³]

Net specific fuel consumption B_{eZ} [g/kWh]

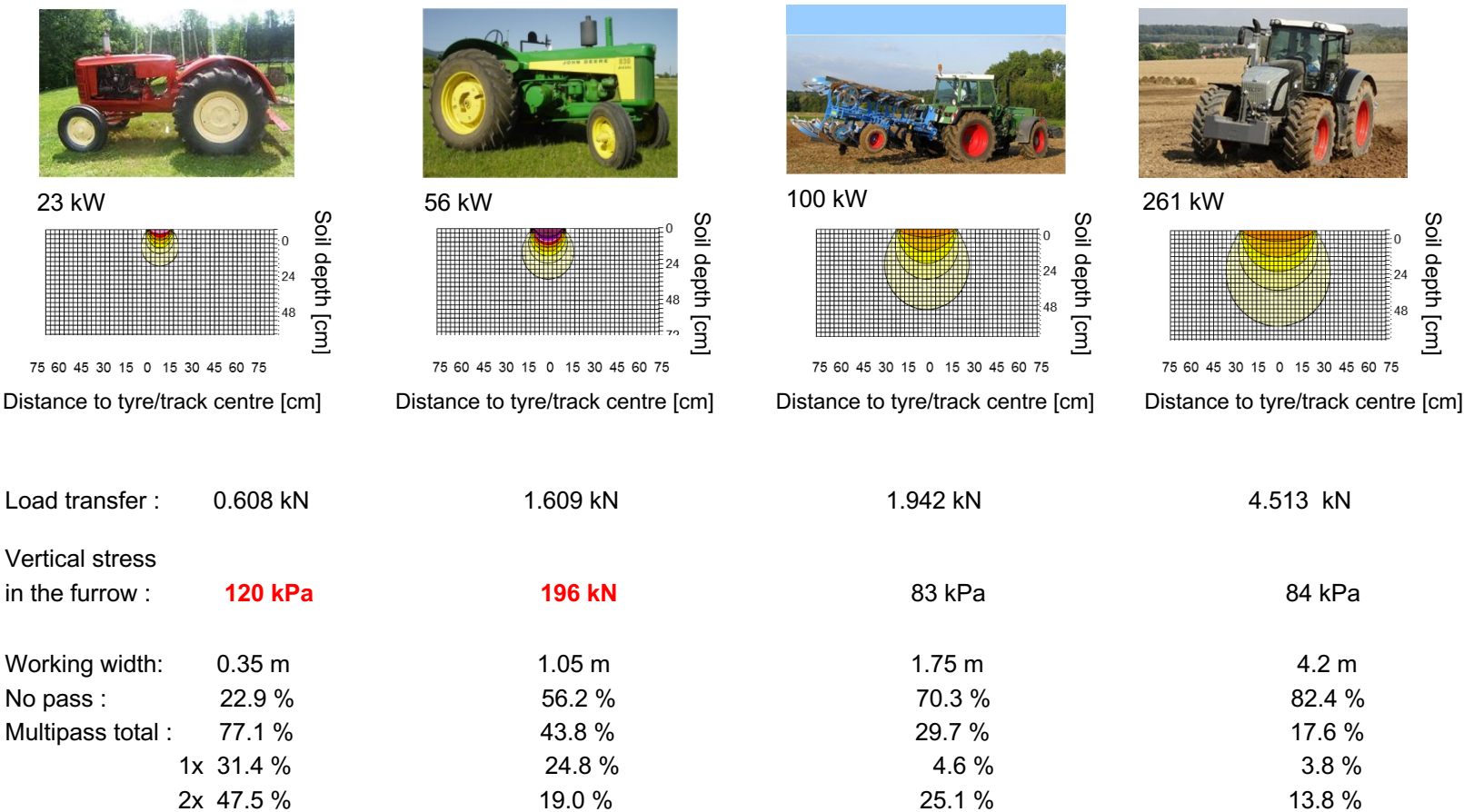
$$b_{eZ} = \frac{P_M \cdot b_{eM}}{P_{NT}}$$

P_{NT} : tractive power [kW]



Evaluation of topsoil damages with TASC

Load transfer, compressive stress at the ploughing pan and pass distribution - silty soil, semi-firm - ploughing

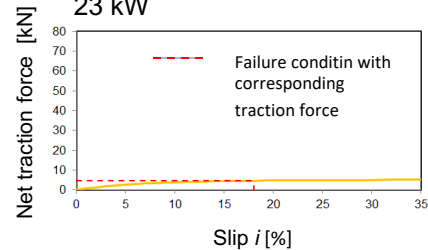


Soil failure risks and fuel consumption with TASC

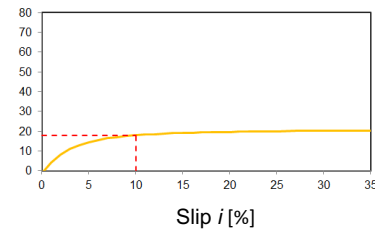
silty soil, semi-firm – plough/ disk harrow – 4 km/h



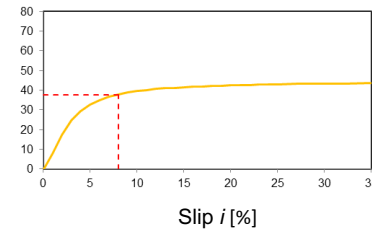
23 kW



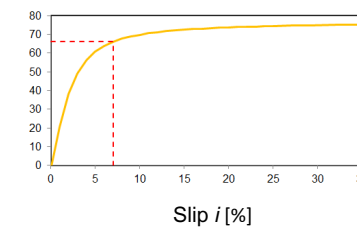
56 kW



100 kW



261 kW



Failure condition : 18 % slip
 Net traction: 4.6 kN
 Draft requirement: 4.5 kN
 Corresponding slip: 16 %
 Plowshare 25 cm 1
 Hourly consumption: 2.2 l/h
 Consumption per hectare: 15.7 l/ha
 Disk harrow offset 15 cm 0.7 m
 Hourly consumption: 1.9 l/h
 Consumption per hectare: 6.8 l/ha

10 % slip
 18.0 kN
 13.5 kN
 4 %
 3
 5.4 l/h
 12.9 l/ha
 3.0 m
 7.4 l/h
 6.2 l/ha

8 % slip
 37.8 kN
 22.5 kN
 3 %
 5
 9.4 l/h
 13.4 l/ha
 6.0 m
 13.6 l/h
 5.7 l/ha

7 % slip
 66.0 kN
 53.9 kN
 4 %
 12
 21.0 l/h
 12.5 l/ha
 10 m
 21.0 l/h
 5.3 l/ha

